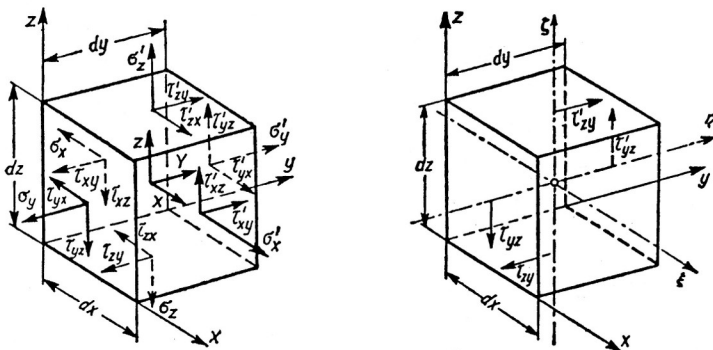


## I. Konstrukcja równowagi geometrycznej (Naviera)

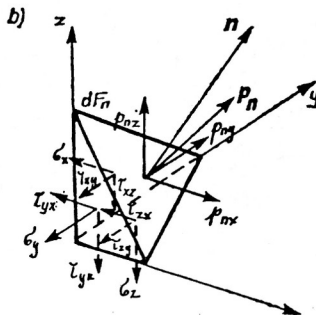
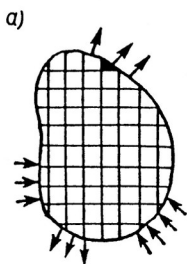


$$\left. \begin{aligned}
 \sigma'_x &= \sigma_x + \frac{\partial \sigma_x}{\partial x} dx, & \tau'_{xy} &= \tau_{xy} + \frac{\partial \tau_{xy}}{\partial x} dx, & \tau'_{xz} &= \tau_{xz} + \frac{\partial \tau_{xz}}{\partial x} dx; \\
 \tau'_{yz} &= \tau_{yz} + \frac{\partial \tau_{yz}}{\partial y} dy, & \sigma'_y &= \sigma_y + \frac{\partial \sigma_y}{\partial y} dy, & \tau'_{yx} &= \tau_{yx} + \frac{\partial \tau_{yx}}{\partial y} dy; \\
 \tau'_{zx} &= \tau_{zx} + \frac{\partial \tau_{zx}}{\partial z} dz, & \tau'_{zy} &= \tau_{zy} + \frac{\partial \tau_{zy}}{\partial z} dz, & \sigma'_z &= \sigma_z + \frac{\partial \sigma_z}{\partial z} dz.
 \end{aligned} \right\}$$

$$\begin{aligned}
 \tilde{\tau}_{xy} &= \tilde{\tau}_{yx} \\
 \tilde{\tau}_{yz} &= \tilde{\tau}_{zy} \\
 \tilde{\tau}_{zx} &= \tilde{\tau}_{xz}
 \end{aligned}$$

$$\left. \begin{aligned}
 \frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} + \frac{\partial \tau_{xz}}{\partial z} + X &= 0, \\
 \frac{\partial \tau_{yx}}{\partial x} + \frac{\partial \sigma_y}{\partial y} + \frac{\partial \tau_{yz}}{\partial z} + Y &= 0, \\
 \frac{\partial \tau_{zx}}{\partial x} + \frac{\partial \tau_{zy}}{\partial y} + \frac{\partial \sigma_z}{\partial z} + Z &= 0.
 \end{aligned} \right\}$$

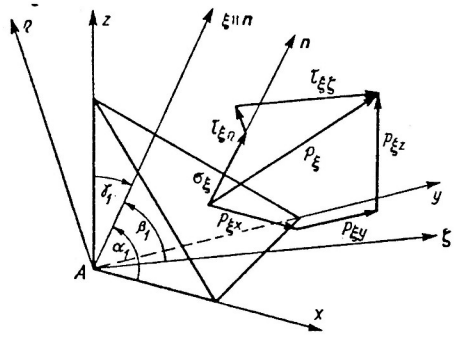
## II. Wzajemki bregowe



$$\begin{aligned}
 a &= \cos(\vec{n}, \vec{i}_x), & dF_x &= a dF_n \\
 b &= \cos(\vec{n}, \vec{i}_y), & dF_y &= b dF_n \\
 c &= \cos(\vec{n}, \vec{i}_z), & dF_z &= c dF_n
 \end{aligned}$$

$$\begin{aligned}
 p_{nx} &= \sigma_x a + \tilde{\tau}_{xy} b + \tilde{\tau}_{xz} c \\
 p_{ny} &= \tilde{\tau}_{yx} a + \sigma_y b + \tilde{\tau}_{yz} c \\
 p_{nz} &= \tilde{\tau}_{zx} a + \tilde{\tau}_{zy} b + \sigma_z c
 \end{aligned}$$

### III Transformacja naprężenia



$$\bar{P}_\xi = \bar{\sigma}_\xi + \tilde{\tau}_{\xi\eta} + \tilde{\tau}_{\xi\zeta}$$

$$\bar{P}_\xi = \bar{P}_{\xi x} + \bar{P}_{\xi y} + \bar{P}_{\xi z}$$

$$\bar{\sigma}_\xi = \bar{P}_{\xi x} a_1 + \bar{P}_{\xi y} b_1 + \bar{P}_{\xi z} c_1 \quad \text{gdzie} \quad a_1 = \cos(\xi x)$$

$$b_1 = \cos(\xi y)$$

$$c_1 = \cos(\xi z)$$

$$\bar{\sigma}_\xi = \bar{\sigma}_x a_1^2 + \bar{\sigma}_y b_1^2 + \bar{\sigma}_z c_1^2 + 2\tilde{\tau}_{xy} a_1 b_1 + 2\tilde{\tau}_{yz} b_1 c_1 + 2\tilde{\tau}_{zx} c_1 a_1$$

$$\tilde{\tau}_{\xi\eta} = \bar{\sigma}_x a_1 a_2 + \bar{\sigma}_y b_1 b_2 + \bar{\sigma}_z c_1 c_2 + \tilde{\tau}_{xy}(a_1 b_2 + a_2 b_1) + \tilde{\tau}_{yz}(b_1 c_2 + b_2 c_1) + \tilde{\tau}_{zx}(c_1 a_2 + c_2 a_1)$$

$$\text{gdzie: } a_2 = \cos(\eta x), \quad b_2 = \cos(\eta y), \quad c_2 = \cos(\eta z)$$

### IV Tensor naprężenia

$$\begin{aligned} \bar{\sigma}_x &= t_{11} & \tilde{\tau}_{xy} &= t_{12} & \tilde{\tau}_{xz} &= t_{13} \\ \tilde{\tau}_{yx} &= t_{21} & \bar{\sigma}_y &= t_{22} & \tilde{\tau}_{yz} &= t_{23} \\ \tilde{\tau}_{zx} &= t_{31} & \tilde{\tau}_{zy} &= t_{32} & \bar{\sigma}_z &= t_{33} \\ \bar{\sigma}_\xi &= t'_{11} & \tilde{\tau}_{\xi\eta} &= t'_{12} & \tilde{\tau}_{\xi\zeta} &= t'_{13} \\ \tilde{\tau}_{\xi\eta} &= t'_{21} & \bar{\sigma}_\eta &= t'_{22} & \tilde{\tau}_{\eta\zeta} &= t'_{23} \\ \tilde{\tau}_{\xi\zeta} &= t'_{31} & \tilde{\tau}_{\zeta\eta} &= t'_{32} & \bar{\sigma}_\zeta &= t'_{33} \end{aligned}$$

$$\begin{aligned} x &= x_1 & \xi &= x'_1 \\ y &= x_2 & \eta &= x'_2 \\ z &= x_3 & \zeta &= x'_3 \end{aligned}$$

	$x_1$	$x_2$	$x_3$
$x'_1$	$a_{11}$	$a_{12}$	$a_{13}$
$x'_2$	$a_{21}$	$a_{22}$	$a_{23}$
$x'_3$	$a_{31}$	$a_{32}$	$a_{33}$

stosując pomysł o oszacowaniu wartości wprost w trzech kierunkach transformacyjnych zastąpić jedynkami o ogólniej postaci:

$$t'_{kl} = a_{ki} a_{lj} t_{ij} \quad (i, j, k, l = 1, 2, 3)$$

wpkazaniu kł odnosi się do nowego układu, natomiast ij do układu pierwotnego.

$$(t_{ij}) = \begin{pmatrix} t_{11} & t_{12} & t_{13} \\ t_{21} & t_{22} & t_{23} \\ t_{31} & t_{32} & t_{33} \end{pmatrix} \rightarrow (t'_{kl}) = \begin{pmatrix} t'_{11} & t'_{12} & t'_{13} \\ t'_{21} & t'_{22} & t'_{23} \\ t'_{31} & t'_{32} & t'_{33} \end{pmatrix}$$

Macierze transformujące się wg poniższego wzoru są tensorami II rzędu.

Jeżeli  $t_{ij} = t_{ji}$  oraz  $t'_{kl} = t'_{lk}$

to są to tensorowy symetryczne składowe stanu napięcia tworzą tensor symetryczny II rzędu.

$$T_{\sigma} = \begin{pmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{pmatrix}$$

V Napięcia główne

Porzucając wartości  $a, b, c$  dla ekstremalnych napięć  $\sigma_{\xi}$  z warunkiem powrotu  $a^2 + b^2 + c^2 = 1$  ustawiamy następujące równanie

$$\frac{\partial F}{\partial a} = 0, \quad \frac{\partial F}{\partial b} = 0, \quad \frac{\partial F}{\partial c} = 0$$

gdzie  $F(a, b, c) = \sigma_{\xi} + \lambda q$   
 $\lambda$  - mnożnik Lagrange'a  
 $q = a^2 + b^2 + c^2 - 1 = 0$

$$\sigma_{\xi}^3 - \Pi_1 \sigma_{\xi}^2 + \Pi_2 \sigma_{\xi} - \Pi_3 = 0$$

gdzie :

$$\begin{aligned} \Pi_1 &= \sigma_x + \sigma_y + \sigma_z \\ \Pi_2 &= \begin{vmatrix} \sigma_x & \tau_{xy} \\ \tau_{yx} & \sigma_y \end{vmatrix} + \begin{vmatrix} \sigma_y & \tau_{yz} \\ \tau_{zy} & \sigma_z \end{vmatrix} + \begin{vmatrix} \sigma_z & \tau_{zx} \\ \tau_{xz} & \sigma_x \end{vmatrix} = \\ &= \sigma_x \sigma_y + \sigma_y \sigma_z + \sigma_z \sigma_x - \tau_{xy}^2 - \tau_{yz}^2 - \tau_{zx}^2 \end{aligned}$$

$$\Pi_3 = \begin{vmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{vmatrix} = \sigma_x \sigma_y \sigma_z + 2 \tau_{xy} \tau_{yz} \tau_{zx} - \sigma_y \tau_{xz}^2 - \sigma_z \tau_{xy}^2 - \sigma_x \tau_{yz}^2$$

$\Pi_1, \Pi_2, \Pi_3$  - niezmienniki stanu napięcia.

$$T_{\sigma} = \begin{pmatrix} \sigma_I & 0 & 0 \\ 0 & \sigma_{II} & 0 \\ 0 & 0 & \sigma_{III} \end{pmatrix}$$

$$\sigma_I > \sigma_{II} > \sigma_{III}$$

